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Amendments to the Specification

Please revise the paragraph beginning on page 1, line 19 of the specification as follows:

In a reciprocating compressor of the related art, a crank shaft 3 is rotated at a certain number of revolution by rotating force of the rotor and the stator which are power transmitting components. And, the The rotational motion is converted into reciprocal straight line motion by a sleeve 16 and a connecting rod 17. A piston 19 is fastened to the connecting rod 17, and moves back and forth within the cylinder 7. By rotational back and forth motion of the piston 19, coolant is inhaled through the suction valve 20 and discharged through the discharging valve 26 after being compressed under the high pressure.

Please revise the paragraph beginning on page 3, line 7 of the specification as follows:

Also, when the piston 19 moves from the bottom dead point to the top dead point, internal pressure of the cylinder 7 is successively elevated to compress the coolant. When such internal pressure of the cylinder 7 becomes larger than spring force of the discharging valve 26, the discharging discharge valve 26 is opened to form a passage through which the high pressure of the coolant is discharged from the cylinder.

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Please revise the paragraph beginning on page 3, line 12 of the specification as follows:

As shown above, when the rotor revolves for one time the operation of the

suction valve 20 and the discharge valve 26 initiated is initiated and suction

and discharge of coolant into and from the cylinder 7 is carried out. When the

suction and discharge valves 20 and 26 are closed, the suction and discharge

valves 20 and 26 collide into the valve plate 22 thereby generating noise and

vibration. The loudness of the collision noise depends on a vibration

transmission capability of the valve plate 22.

Please revise the paragraph beginning on page 7, line 7 of the

specification as follows:

Hereinafter detailed a detailed description will be made about of the

embodiments of the invention is provided in reference to FIG. 3 EIGS. 3A to

FIG. 10, in which some components of the invention are designated with the

same reference numerals as those of the related art for the convenience's sake

of explanation.

Please revise the paragraph beginning on page 7, line 16 of the

specification as follows:

As shown in FIG. 4, vibration and noise generated from collision of the

suction valve 20 and the discharging discharge valve 26 into the valve plate 22

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spreads into all directions about a suction port. Noise and vibration are transmitted to the grooves composed of the number of cavities 118 during a spreading process. The cavities 118 are hollow spaces with various widths and respectively have the resonant frequencies pertinent thereto so that only the pertinent frequencies are resonated in the cavities 118, that is, the cavities 118 obstruct transmission of vibration and noise with the frequencies pertinent to the resonant frequencies to damp the amplitude of vibration and noise.

Please revise the paragraph beginning on page 7, line 24 of the **specification** as follows:

According to such a principle, when the cavities are adjusted in width in correspond to the most problematic bands of vibration and noise generated from collision of the suction and discharging discharge valves 20 and 26 with the valve plate 22.